

Planar Dynamics

Engineering Mechanics: Dynamics

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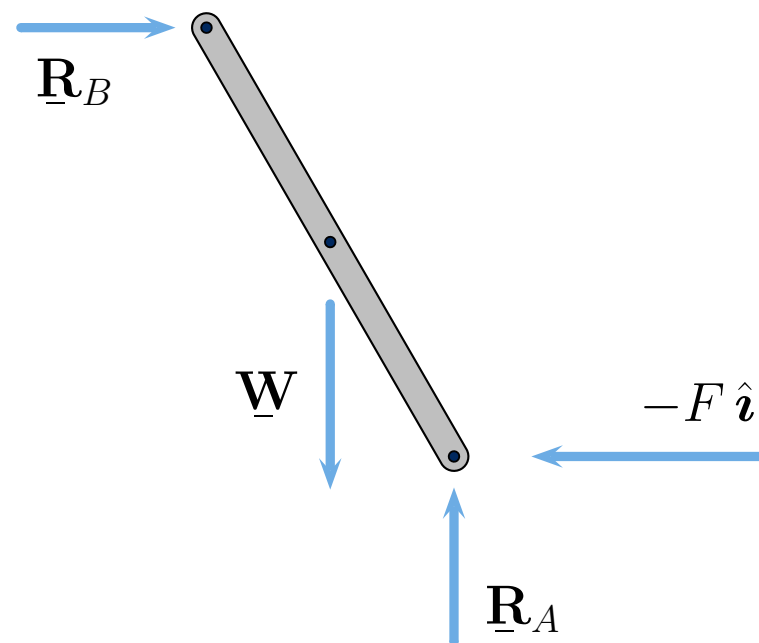
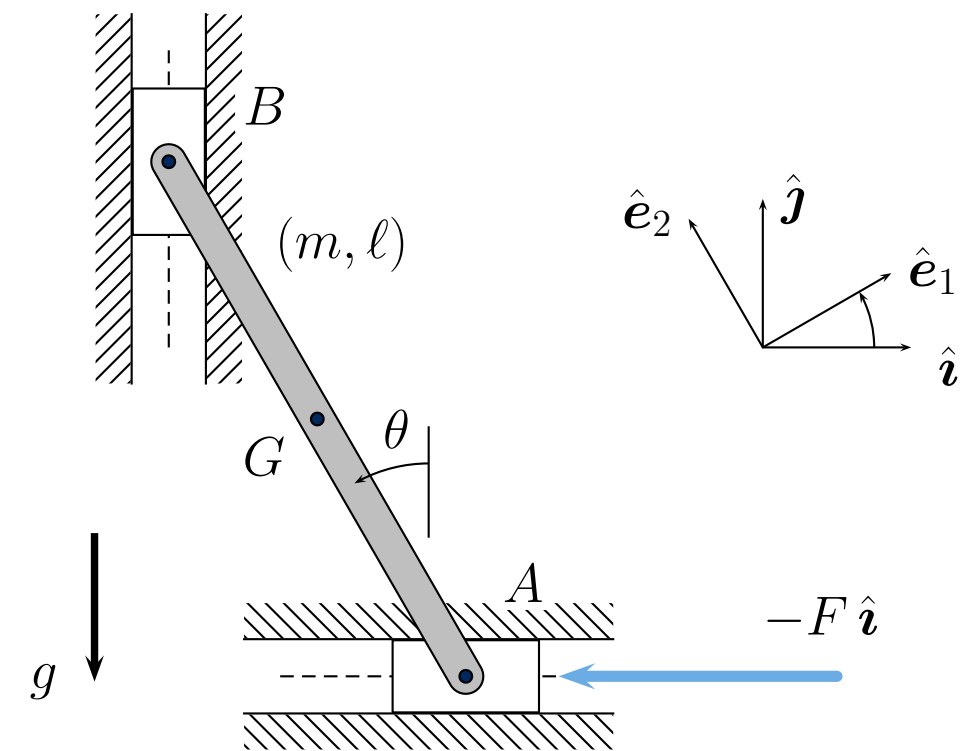
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The bar of mass m and length ℓ slides along the smooth horizontal and vertical slots and the (massless) block at A is subject to a force $-F \hat{i}$. Find the resulting angular acceleration of the bar if it is released from rest at angle θ .



- The forces acting on the bar arise from
- ▶ Reaction forces (\underline{R}) on the blocks
 - ▶ Weight (\underline{W}) of the bar
 - ▶ External force ($-F \hat{i}$)

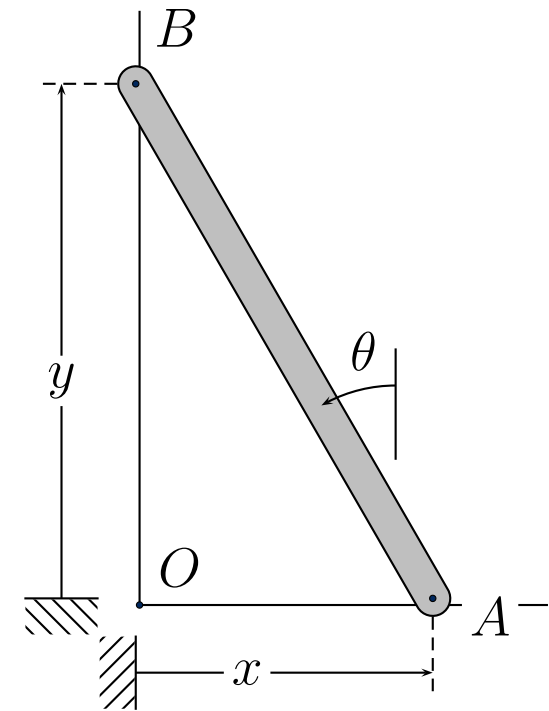
Coordinates and Directions

Define the directions (\hat{e}_1, \hat{e}_2) fixed in the bar and (\hat{i}, \hat{j}) fixed in the ground, so that

$$\hat{e}_1 = C_\theta \hat{i} + S_\theta \hat{j}, \quad \hat{e}_2 = -S_\theta \hat{i} + C_\theta \hat{j},$$

where the rotation of the bar is defined by θ . Also, the blocks are constrained to move along the slots, so that

$$\underline{\mathbf{r}}_{A/O} = x \hat{i}, \quad \underline{\mathbf{r}}_{B/O} = y \hat{j}.$$



Kinematics

Because of the sliding constraints this is a single degree-of-freedom system, so that the motion of the mass center can be related to the rotation of the bar. Note that

$$\underline{\mathbf{a}}_B = \underline{\mathbf{a}}_A + \underline{\boldsymbol{\alpha}}_B \times \underline{\mathbf{r}}_{B/A} + \underline{\boldsymbol{\omega}}_B \times (\underline{\boldsymbol{\omega}}_B \times \underline{\mathbf{r}}_{B/A}),$$

$$\ddot{y} \hat{j} = \ddot{x} \hat{i} + (\ddot{\theta} \hat{k}) \times (l \hat{e}_2) + \underline{\mathbf{0}} = \ddot{x} \hat{i} - l \ddot{\theta} \hat{e}_1, \quad \longrightarrow \quad \hat{i} \text{ direction: } 0 = \ddot{x} - l \ddot{\theta} C_\theta,$$

$$\hat{j} \text{ direction: } \ddot{y} = -l \ddot{\theta} S_\theta.$$

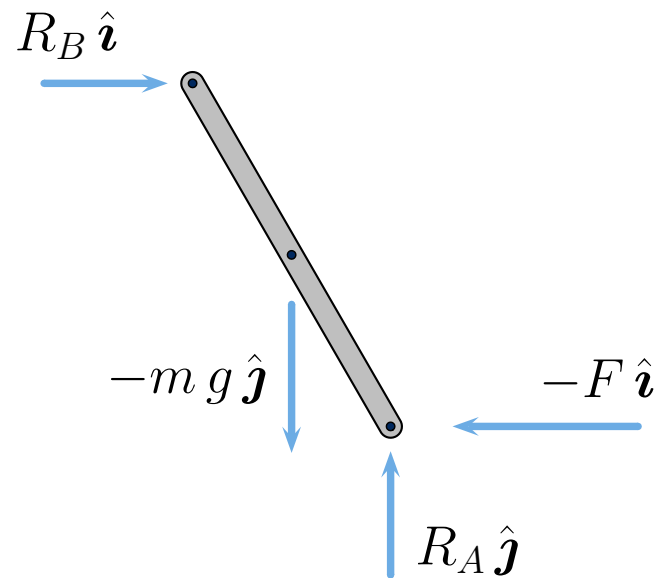
Therefore

$$\ddot{x} = l \ddot{\theta} C_\theta, \quad \ddot{y} = -l \ddot{\theta} S_\theta$$

Finally,

$$\begin{aligned}\underline{\mathbf{a}}_G &= \underline{\mathbf{a}}_A + \underline{\boldsymbol{\alpha}}_B \times \underline{\mathbf{r}}_{G/A} + \underline{\boldsymbol{\omega}}_B \times (\underline{\boldsymbol{\omega}}_B \times \underline{\mathbf{r}}_{G/A}), \\ &= \ddot{x} \hat{\mathbf{i}} + (\ddot{\theta} \hat{\mathbf{k}}) \times \left(\frac{\ell}{2} \hat{\mathbf{e}}_2 \right) + \mathbf{0}, \\ &= \left(\ell \ddot{\theta} C_\theta \right) \hat{\mathbf{i}} - \frac{\ell}{2} \ddot{\theta} \hat{\mathbf{e}}_1 = \left(\frac{\ell}{2} \ddot{\theta} C_\theta \right) \hat{\mathbf{i}} + \left(-\frac{\ell}{2} \ddot{\theta} S_\theta \right) \hat{\mathbf{j}}.\end{aligned}$$

Free Body Diagram/Equations of Motion



Momentum balance for the bar can be written as

$$\begin{aligned}\sum \underline{\mathbf{F}} &= (R_B - F) \hat{\mathbf{i}} + (R_A - m g) \hat{\mathbf{j}} \\ &= m \left(\left(\frac{\ell}{2} \ddot{\theta} C_\theta \right) \hat{\mathbf{i}} + \left(-\frac{\ell}{2} \ddot{\theta} S_\theta \right) \hat{\mathbf{j}} \right) = m \underline{\mathbf{a}}_G,\end{aligned}$$

$$\sum \underline{\mathbf{M}}_G = \left(\frac{\ell}{2} S_\theta R_A - \frac{\ell}{2} C_\theta (F + R_B) \right) \hat{\mathbf{k}} = \frac{m \ell^2}{12} \ddot{\theta} \hat{\mathbf{k}} = I_G \underline{\boldsymbol{\alpha}}_B.$$

Taking components of these in the $(\hat{i}, \hat{j}, \hat{k})$ directions

$$R_B - F = \frac{m \ell C_\theta}{2} \ddot{\theta}, \quad R_A - m g = -\frac{m \ell S_\theta}{2} \ddot{\theta}, \quad \frac{\ell S_\theta}{2} R_A - \frac{\ell C_\theta}{2} (F + R_B) = \frac{m \ell^2}{12} \ddot{\theta}.$$

Therefore

$$R_A = m g - \frac{m \ell S_\theta}{2} \ddot{\theta}, \quad R_B = F + \frac{m \ell C_\theta}{2} \ddot{\theta},$$

$$\longrightarrow \frac{\ell S_\theta}{2} \left(m g - \frac{m \ell S_\theta}{2} \ddot{\theta} \right) - \frac{\ell C_\theta}{2} \left(F + F + \frac{m \ell C_\theta}{2} \ddot{\theta} \right) = \frac{m \ell^2}{12} \ddot{\theta},$$

so that

$$\ddot{\theta} = \frac{3}{m \ell} \left(\frac{m g S_\theta}{2} - F C_\theta \right).$$

Also, we can solve for the reactions as

$$R_A = \left(1 - \frac{3 S_\theta^2}{4} \right) m g + \frac{3 S_\theta C_\theta}{2} F,$$

$$R_B = \frac{3 S_\theta C_\theta}{4} m g + \left(1 - \frac{3 C_\theta^2}{2} \right) F.$$

